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## ME 1110 – Engineering Practice 1

### Engineering Drawing and Design - Lecture 14

# Mechanical Elements – Bearings

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[www.staff.city.ac.uk/~ra600/intro.htm](http://www.staff.city.ac.uk/~ra600/intro.htm)

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## Introduction

### What is a bearing?

- In general - a support or supporting element
- In machine design - a component that allows for relative motion between two bodies

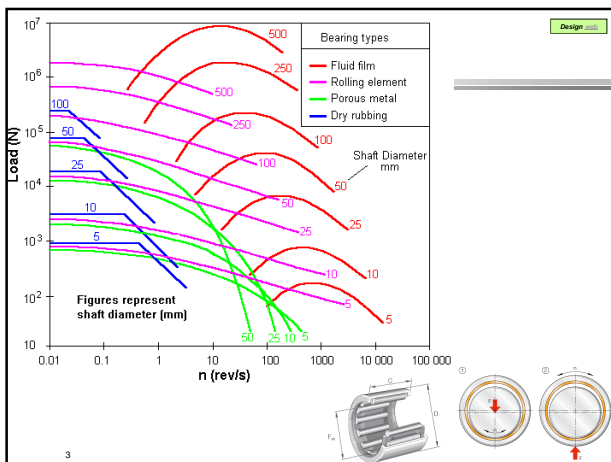
Your skeleton is the central structure that supports your body and its modules  
 Your **joints are bearings** that allow different body modules to move with respect to each other

- Bearings allow machines or their parts to move

Bearings can have many forms, but only two types of motions:  
**Linear** motion or **rotary** motion

- In all bearings, cleanliness and surface finish are most important
- There are many different types of bearings:
  - Sliding
  - Rolling**
  - Flexing
  - Fluid Film (hydrodynamic)

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## Rolling element bearings

Labels in the diagram include: Width, Outer ring, Inner ring, Bore, Face, Corner radius, Shoulders, Inner ring ball race, Separator (retainer), and Outer ring ball race.

Designed to take:

- Pure radial loads
- Pure thrust loads
- Combination of the two kinds of loads

Main parts:

- Outer ring
- Inner ring
- Rolling elements (balls)
- Separator

Selection of bearings:

- Type and amount of load (axial - thrust, radial)
- Size, Speed
- Lubrication
- Life rating

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## Bearing assembly

- The inner ring inserted into the outer ring
- Balls inserted in the remaining space
- Balls distributed equidistantly, rings concentric
- Retainer secures the balls

Number and size of balls determine the bearing load

- More balls - smaller balls > higher load-lower speed
- Less balls - bigger balls > lower load-higher speeds

Rolling bearing types:

- Ball - roller
- Needle - taper roller & spherical roller

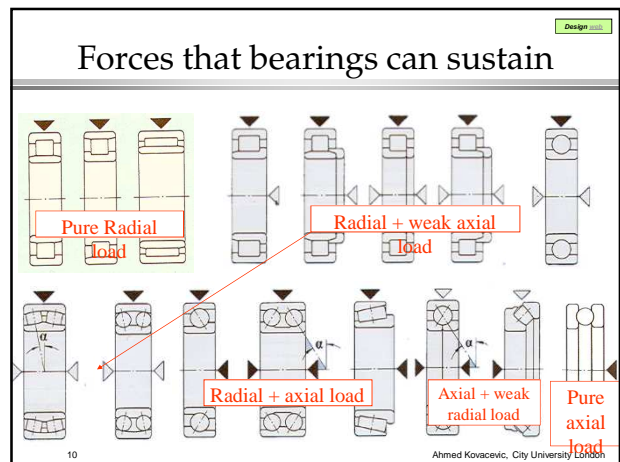
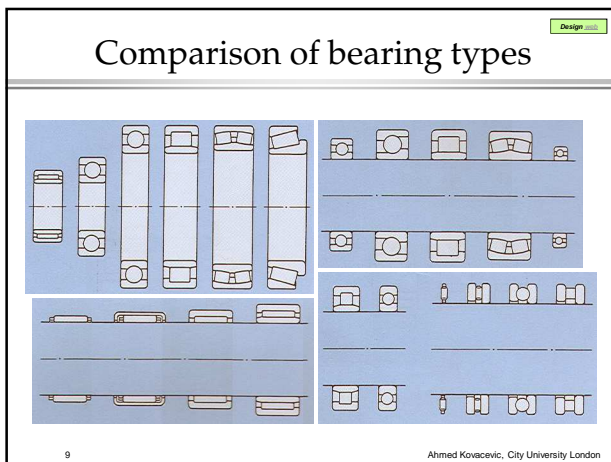
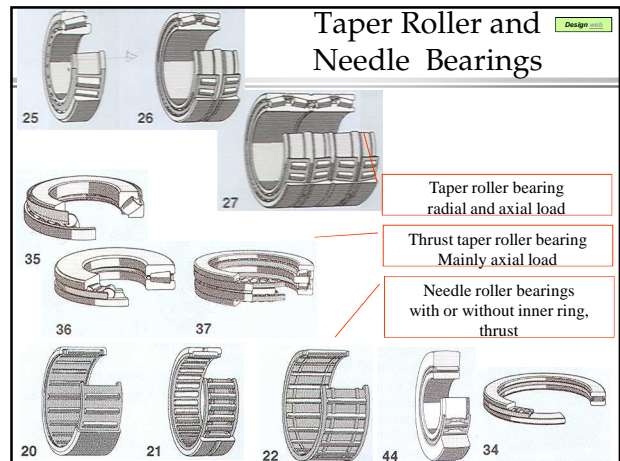
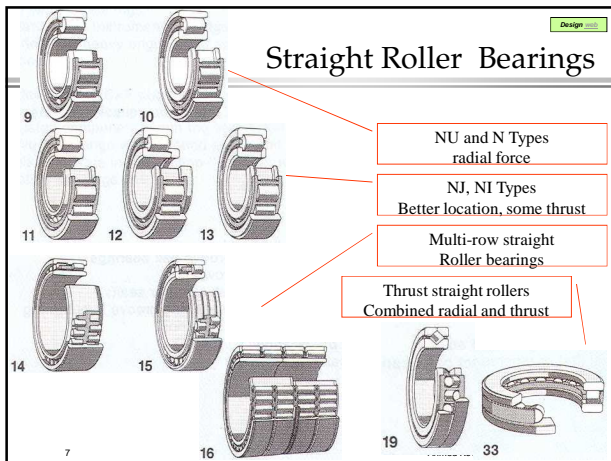
Roller bearings take more load than ball bearings

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## Ball Bearings

- Deep groove ball bearings: Take both radial and axial force
- Angular contact - axial force: 2 point; 4 point
- Thrust Ball bearings: Deep groove, angular contact
- Special, self-aligning and combined Ball bearings

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### How to select the bearing from the catalogue

Principal dimensions										Basic load ratings		Fatigue load limit		Speed ratings		Mass		Designation		
d	D	B	C	C <sub>2</sub>	r <sub>1</sub>	r <sub>2</sub>	r <sub>3</sub>	r <sub>4</sub>	r <sub>5</sub>	N	C <sub>10</sub>	P <sub>0</sub>	P <sub>063</sub>	n	n <sub>1</sub>	n <sub>2</sub>	m	m <sub>2</sub>	Designation	
mm	mm	mm	mm	mm	mm	mm	mm	mm	mm		kN	kN	kN	1/min	1/min	1/min	kg	kg		
36	47	7	4.256	3.000	386	11.000	18.000	0.030	0.030	61807				35	36.7	43.5	0.3	37	45	0.3
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36	47	7	4.256	3																

## Bearing Life - Calculation

- The **size** of a bearing is initially selected on the relation of its **load carrying capacity** and the carried **load** with the **life and reliability** requirement.
- Load carrying capacity** is specified for each bearing in a catalogue with:
  - C – basic dynamic load rating – for variable loads and high speeds
  - C<sub>0</sub> – basic static load rating – for static loads and low speeds
- The load calculated from free body diagrams or by other means.
- Rating life** can be calculated from the life equation. The form of the life equation depends on the accuracy required. **Basic rating life is:**

$$L_{10} = \left(\frac{C}{P}\right)^a \quad [10^6 \text{ rev}]$$

$$L_{10h} = \frac{10^6}{60n} L_{10} \quad [\text{hours}]$$

$$L_{10s} = \frac{\pi D}{1000} L_{10} \quad [10^6 \text{ km}]$$

a = 3 – for ball bearings  
 a = 3.33 – for roller bearings  
 P [N] – equivalent dynamic load rating  
 n [rpm] – rotational speed  
 D [m] – wheel diameter

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## Adjusted Bearing Life

- If a bearing is not operating in the ideal conditions then the basic rating life should be adjusted:

$$L_{adj} = a_T a_R a_{OC} L_{10} \quad [10^6 \text{ rev}]$$

Temperature [°C]	150	200	250	300		
a <sub>T</sub>	1.00	0.90	0.75	0.60		
Reliability [%]	90	95	96	97	98	99
a <sub>R</sub>	1.00	0.62	0.53	0.44	0.33	0.21

- Coefficient a<sub>OC</sub> represents means of lubrication and dust prevention. It varies from 0.20 – 2.20. Values depend on relative viscosity of lubricant. Values higher the one are possible for special lubricants and cooling.

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## Equivalent dynamic bearing load

$$P = x F_r + y F_a$$

P [N] – equivalent dynamic bearing load  
 F<sub>r</sub> [N] – actual radial bearing load  
 F<sub>a</sub> [N] – actual axial bearing load  
 x – radial load factor  
 y – axial load factor

Bearing type	Condition	x	y
Deep groove ball bearing	F <sub>d</sub> /F <sub>r</sub> ≤ 0.5	1	0
	F <sub>d</sub> /F <sub>r</sub> > 0.5	0.56	1-2
Self-aligning ball bearings	F <sub>d</sub> /F <sub>r</sub> ≤ e*	1	Y*
	F <sub>d</sub> /F <sub>r</sub> > e*	0.65	Y*
Angular contact ball bearings	F <sub>d</sub> /F <sub>r</sub> ≤ 1.14	1	0
	F <sub>d</sub> /F <sub>r</sub> > 1.14	0.35	0.57
Double row angular contact ball bearings	F <sub>d</sub> /F <sub>r</sub> ≤ 0.86	1	0.73
	F <sub>d</sub> /F <sub>r</sub> > 0.86	0.62	1.17
Four-point contact ball bearings	F <sub>d</sub> /F <sub>r</sub> ≤ 0.95	1	0.66
	F <sub>d</sub> /F <sub>r</sub> > 0.95	0.6	1.07
Cylindrical roller bearing (with flanges)	F <sub>d</sub> /F <sub>r</sub> ≤ 0.2	1	0
	F <sub>d</sub> /F <sub>r</sub> > 0.2	0.92	0.6
Needle roller bearings	-	1	0
Trust roller bearings	-	0	1
Taper roller bearings	F <sub>d</sub> /F <sub>r</sub> ≤ e*	1	0
	F <sub>d</sub> /F <sub>r</sub> > e*	0.4	Y*
Taper roller bearings	1.00	0.75	0.60

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Bearing Type	Direction of Load			Ratio of Load/Bulk			Misalignment Capacity		
	radial	axial	both	high	med	low	high	med	low
Thrust Ball		y			y				y
Deep Groove Ball	y				y			y	
Cylindrical Roller	y		certain types		y				y
Needle Roller	y				y				y
Tapered Roller	y	y	y		y				y
Self-aligning Ball	y					y	y		
Self-aligning Spherical Roller	y				y		y		
Angular Contact Ball		y	y			y			y

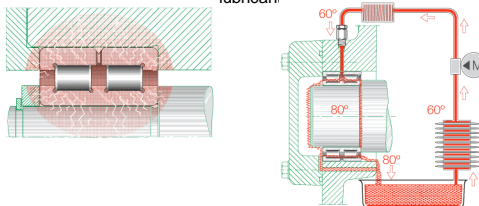
Machine Usage Type	Life Required of Bearings (Hours)
household appliances – intermittent use	300 – 3000
hand tools, construction equipment – short period use	3000 – 8000
lifts, cranes – high reliability for short periods	8000 – 12000
8h/day gears, motors – full day partial use	10000 – 25000
8h/day machine tools, fans – full day full use	20000 – 30000
continuous use	40000 – 50000

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## Bearing speeds

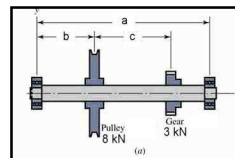
- There is a limit to the speed at which rolling bearings can operate.
- The **top speed** is limited by the operating **temperature** of the bearing. The heat is generated by the **friction** between rolling elements. The heat is removed by: the conduction through the shaft and housing lubricant



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## Example



Select the bearings and determine their rating life for the driving mechanism shown in the Figure. The shaft is 450 mm long and supported by deep-groove bearing in point O and plane roller bearing in point C. Assume minimum shaft diameter to be 20 mm. Mounted upon the shaft are a V-belt pulley, which contributes a radial load of F<sub>1</sub>=8kN to the shaft, and a gear which contributes a radial load of F<sub>2</sub>=3kN. The two loads are in the same plane and have the same direction. Minimum required bearing life is 2000 h with 90% reliability. Shaft rotates constantly at n=1000 rpm.

F<sub>1</sub>=8 kN      a=450 mm      c=200 mm  
 F<sub>2</sub>=3 kN      b=150 mm      d=20 mm  
 L<sub>10h</sub>=(L<sub>10h</sub>)<sub>0}=(L<sub>10h</sub>)<sub>C}=2000 h      n=1000 rpm</sub></sub>

**SOLUTION:**

$$L_{10h} = \frac{10^6}{60n} \left(\frac{C}{P}\right)^a \Rightarrow C = P * \sqrt[6]{\frac{60n}{10^6} L_{10h}}$$

$$C_0 = 6000 * \sqrt[6]{\frac{60 * 1000}{10^6} * 2000} = 29,595 \text{ N}$$

$$C_0 = 5000 * \sqrt[6]{\frac{60 * 1000}{10^6} * 2000} = 21,025 \text{ N}$$

$$P_0 = R_1 = 6000 \text{ N} \quad P_C = R_2 = 5000 \text{ N}$$

Selected from the catalogue for deep-groove ball bearings:  
 6404 20x72x19 mm C=30,700 N

Selected from the catalogue for cylindrical roller bearings:  
 NU 204 20x47x14 mm C=25,100 N

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