ME1105 - Engineering Drawing and Design

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1 The design process and the role of the design model.

1.1 The design process:

Almost everything around us has been created by, or is influenced by, engineers:

Buildings, vehicles, roads, railways, food growing and processing, books, medical care, recreation, etc.

All of these have either been concieved and created from scratch or have evolved from existing ideas. Either way, an engineering design process will have been followed, in one form or another. The **Design as a generic tool** module provides an interesting a comprehensive introduction to engineering and design, so a detailed discussion of the design process will not be inclided here.

In essence, designs progress from :

some statement of need to.. identification or specification of problem to.. search for solutions and finally to... development of solution to manufacture, test and use.

This sequence is usually iterative. It repeats until a satisfactory solution has evolved, as indicated in the flow diagram below.



1.2 The design model.

The concept of the designer working with a **model** of a design is fundamental to the design process.

The design **model** is a representation of the design. This model could be anything from a few ideas in the designers head, through to rough sketches and notes, calculations, sets of detailed formal engineering drawings, computer generated 3D representations, physical prototypes, etc.

The design model would be used by the designer to record and develop ideas and to provide a basis to evaluate the design.

Larger design projects are undertaken by more than one engineer. Design models are used to communicate and demonstrate ideas between all those concerned with the product design, development, manufacture and use.

A designer needs to have the skills to generate and work with this model in order to communicate ideas and develop a design.

<u>1.3 Types of design model.</u>

Designers use a variety of different models, depending on what property of the design is to be considered and for whom the information is destined.

Typically a designer may model:

- Function
- Structure
- Form
- Material properties, surface conditions

All of these areas probably encompass a large portion of the degree syllabus. Within this modulke we will concern ourselves primarily with form, i.e. the shape of parts or components and how they fit together.

2.1 Projections.

2.1.1 Orthographic projection.

We have discussed both the role of the **design model** in the design process and the importance of the **representation of the form** or shape in this role.

Now we will consider in detail the methods designers use to **represent the form** of their designs.

Back in the 18th century a French mathematician and engineer, Gaspard Monge (1746-1818), was involved with the design of military armory. He developed a system, using two planes of projection at right angles to each other, for graphical description of solid objects.



Figure 2.1a, two right angle planes of projection.

ORTHOS -GRAPHOS - straight, rectangular, upright written, drawn

Orthographic projection is the graphical method used in modern engineering drawing. In order to interpret and communicate with engineering drawings a designer must have a sound understanding of it's use and a clear vision of how the various projections are created.

There are two predominant **orthographic projections** used today. They are based on Monge's original right angle planes and are shown fully in Figure 2.1b. They define four separate spaces, or quadrants. Each of these quadrants could contain the object to be represented. Traditionally however, only two are commonly used, the **first** and the **third**.



This system, which was, and still is, called **Descriptive Geometry**, provided a method of graphically describing objects accurately and unambiguously. It relied on the perpendicular projection of geometry from perpendicular planes.

Monge's Descriptive Geometry forms the basis of what is now called **Orthographic Projection**.

The word **orthographic** means to draw at right angles and is derived from the Greek words:



Figure 2.1b.

Projections created with the object placed in the first quadrant are said to be in **First Angle** projection, and likewise, projections created with the object placed in the third quadrant are said to be in **Third Angle** projection.

2.1.2 First angle projection.

Consider the first quadrant in Figure 2.1b. The resultant drawing of the cone would be obtained by flattening the two perpendicular projections planes, as shown in Figure 2.1c.



For this example, you could say that the right hand side image is the plan or **top** elevation and the image to the left is the **side elevation**.

Whether you view the objects from the left or the right, the order in which the drawing views are arranged puts the image that you see **after** the **object**, **object first then the image**. This is always true for **First Angle** projection.

Put another way:

- Viewing from the left: The drawn image on the right is your view of the drawn object on the left.
- Viewing from the right: The drawn image on the left is your view of the drawn object on the right.

This can get confusing, particularly when also considering other drawings created using other projections. You may develop your own way of recognising First Angle projection. The author uses:

The **OBJECT** is **FIRST** for **FIRST** Angle projection.

or... EYE > OBJECT > IMAGE or... You look through the object and place the image



An example of a component represented in a multiview drawing, in **First Angle** projection.

2.1.3 Third angle projection.

Consider the third quadrant in Figure 2.1b. The resultant drawing of the cone would be obtained by flattening the two perpendicular projections planes, as shown in Figure 2.1d.



Figure 2.1d, Third Angle.

For this example of the cone, you would say that the left hand image is the plan or **top** elevation and the image to the right is the **side elevation**.

Whether you view the objects from the left or the right, the order in which the drawing views are arranged puts the image that you see **before** the **object**, **image first then the object**. This is always true for **Third Angle** projection.

Put another way:

- Viewing from the left: The drawn image on the left is your view of the drawn object on the right.
- Viewing from the right: The drawn image on the right is your view of the drawn object on the left.

Again, you may develop your own way of recognising Third Angle projection. Perhaps: **EYE > IMAGE> OBJECT**



The same component shown using **Third Angle** projection.

2.1.4 Orthographic projection symbols.

Both systems of projection, First and Third angle, are approved internationally and have equal status. The system used must be clearly indicated on every drawing, using the appropriate symbol shown in Figure 2.1e below.



Figure 2.1e. Projection system symbols and recommended proportions.

2.1.5 Pictorial Drawing.

Orthographic projection is used as an unambiguous and accurate way of providing information, primarily for manufacturing and detail design. This form of representation can however make it difficult to visualise objects. Pictorial views can be created to give a more three dimensional impression of the object. There are three types of pictorial projections commonly used, as shown in Figure 2.1f.



Figure 2.1f. Perspective, isometric and oblique pictorial projections.

Perspective: Used more with freehand sketching.

Parallel lines appear to converge and meet at what is referred to as the **vanishing point**. You can have one, two or three vanishing points (VP).



Isometric: Receeding lines drawn at 30° and are usually kept at true measured lengths.



Oblique: Front face sketched as a true shape. Starts with two axes, one horizontal, one vertical. The third axis is usually drawn at 45° and lengths are reduced by 50% of true lengths. Sometimes called 'cabinet' projection.



L

R

Ρ

This is an introduction into how to create and interpret multi-view orthographic projection drawings.

2.2.1 First angle projection.

The component:

Your drawing will, for this example consist of four views:

- Front **F**
- Left
- Right
- Plan (Top)
- Usual practice is to orient the component in a position that it is most likely to be found in.



Your aim is to create, from the front view, an orthographic projection drawing as shown below in Figure 2.2a. Note how the views are constructed in line with each other, allowing the features to be 'projected' between the views.



Figure 2.2a. A completed First angle projection drawing.

So, the stages are:

1) Choose which view direction or face will be used as the front view of the component.

2) Draw the outline of the front view, leaving room for the other views.



3) Draw feint construction lines out from the front view.

4) Start to draw the outlines of the other views, using sides you know the length of.



5) Complete the details of the views by adding any required hidden detail lines, other outlines and center lines.

(Refer to section 2.3 for line style conventions.)

With first angle projection the plan view is **below** the front view. If you had placed the plan view **above** the front view it would actually have to

become the bottom or underside view!



2.2.2 Third angle projection.

The construction method used is the same. The difference between first and third angle projection when creating or reading really lies with the positions of the views. For the same component, an orthographic projection drawing with the same front, side and plan views would look like Figure 2.2b below.



Figure 2.2b. Third angle projection.

Observe how, in third angle, the views give the image then the object. In other words, what you see then what you are looking at.

In first angle you are given the object then the image, or what you are looking at, then what you see.

2.3 Drawing conventions.

2.3.1 Introduction.

In order for anyone to be able to understand exactly what a drawing represents, sets of precise rules and conventions have to be followed, much like a language. These rules are usually referred to as **Standards**.

When a designer works to a set of Standards they must be familiar with the precise meaning of the various line styles, abbreviations, drawing simplifications and terminology. This section introduces you to some of the common universal conventions.

Standards are developed both privately by companies and by internationally recognised institutions.

Two such international standards are:

British Standard Institution: **BS 8888** (Superceded BS 308)

American National Standards Institute: Y14 series

The conventions referenced in these notes generally comply with BS 8888.

Each line on a drawing represents specific precise information regarding the components design.

Туре:	Example:	Application:
Continuous	A	Visible outlines
Continuous (thin)	В	Dimension lines
Short dashes	C	Hidden detail
Long chain	D	Center lines
Chain, thick at ends	E	Section cutting planes
Short chain	F	Developed views
Continuous wavy boundaries	G	Broken
Straight zigzag	Н	Break lines
Straight lines with two short zigzags	I	Dimension lines

2.3.3 Terminology & representations of standard components.

Here are some examples of commonly used engineering components and features of components.

General:	
Housing: A component into which a 'male' mating part fits, sits or is 'housed'.	Housing
Bush/bearing: A removable sleeve or liner. Known also as a simple or plane bearing.	Bush (bearing)
Boss: A cylindrical projection on surface of component.	Curved Slot
Curved slot: Elongated hole, whose centerline lies on an arc. Used usually on components requiring adjustment.	Boss
Rib: A reinforcement, positioned to stiffen surfaces.	
Fillet: A radius or rounded portion suppressing a sharp internal corner.	Fillet
Key: A small block or wedge inserted between a shaft and a mating part (a hub). Used to prevent relative rotation of the two parts.	Keyway
Key way: A parallel sided slot or groove cut into a bore or a shaft, to 'house' a mating key.	























Worm & wheel:	Wheel
	wheel in mesh. Worm

Shaft ends:	
Square: Frequently used for hand driven adjustments with removable handles, such as those found on machine tools, etc.	
Serrations: Often used for push fit components such as plastic fans or pulleys, or levers such as motorcycle gear shifters.	



Belt drives: V belt drives: Used for transmission of rotary power, good for space restricted applications. Vbelts grip on the sides of the V. Often found on automotive engines to drive alternators and water pumps, or on pillar drills, and other industrial drives. **Timing or synchronous** drives: Used for transmission of rotary power, as are v-belts, and, because of the toothed design (no slip) they are used for timed (synchronised) drives, where relative rotational positions have to be controlled. Some type of tensioning system is usually required. These drives are often found on camshaft drives on modern automotive engines, replacing chains. + Pitch

Abbreviations are used on drawings to save time and space. Most of these conform to BS 8888. They are the same singular or plural, full stops are only used where word may be confusing.

A/C	Across corners
A/F	Across flats
HEX HD	Hexagon head
ASSY	Assembly
CRS	Centers
CL	Center line
CHAM	Chamfer
CH HD	Cheese head
CSK	Countersunk
CBORE	Counterbore
CYL	Cylinder or cylindrical
DIA	Diameter (in a note)
Ø	Diameter (preceding a dimension)
R	Radius (preceding a dimension, capital only)
RAD	Radius (in a note)
DRG	Drawing
FIG.	Figure
LH	Left hand
LG	Long
MATL	Material
NO.	Number
PATT NO.	Pattern number
PCD	Pitch circle diameter
I/D	Inside diameter
O/D	Outside diameter
RH	Right hand
RD HD	Round head
SCR	Screwed
SPEC	Specification
SPHERE	Spherical
SFACE	Spotface
SQ	Square (in a note)
ТҮР	Typical or typically
ТНК	Thick
	Square (preceding a dimension)
STD	Standard
UCUT	Undercut
M/CD	Machined
mm	Millimeter
NTS	Not to scale
RPM	Revolutions per minute
SWG	Standard wire gauge
TPI	Teeth per inch

2.4 Sections.

To show the inside details of component it is imagined to be cut or sectioned along a plane, the **cutting plane**. **Cutting planes** are designated with capital letters, such as **A-A** in Figure 2.4a.



The side of the plane nearest the viewer is removed

systems are still used and are indicated by use of the

and the remaining details are shown as a sectional view, as demonstrated with section **X-X** in Figure 2.4b. The arrows indicate the direction to view the component when defining the sectioned view. Note that First or Third angle orthographic projection



Figure 2.4b.

Sectional views are produced to:

- clarify details
- show internal features clearly
- reduce number of hidden detail lines required
- aid dimensioning

appropriate symbols.

- show cross-section shape
- clarify an assembly



Surfaces cut by the **cutting plane** are usually hatched at an appropriate angle, say 45° with a density of lines in proportion with the component.

Symmetrical parts can be shown in **half** sections. **Part** or 'broken out' sections can be used.





Figure 2.4c. Half section and a part or 'broken out' section.

Revolved sections are useful when clarifying local cross-section shapes as shown in Figure 2.4d.



Figure 2.4d.

There are some exceptions to the general rules of sectioning:

- Webs, see Figure 2.4e.
- Shafts, rods, spindles, see Figure 2.4f.
- Bolts, nuts and thin washers.
- Rivets, dowels, pins and cotters.

These parts would not be shown as sections if their center lines lie on the cutting plane.





Wrong!

Web is not sectioned.

Section B-B



Section X-X Section Y-Y



Figure 2.4e. Web sections.



It may be appropriate to use **Removed** sections, for webs, beams or arms, as shown in Figure 2.4g below. Note the absence of viewing arrows.



Figure 2.4g. Removed sections.

Assemblies can be greatly clarified using sections. See the example below in Figure xx.

Note:

- Revolved sections.
- Part sections.
- Different hatching directions and spacings.
- Un-sectioned components such as shafts, keys, nuts etc.



Figure 2.4h. An assembly drawing view, clarified using sections.

2.5 Dimensions.

A drawing from which a component is to be manufactured must communicate design information by:

- Describing the form or shape of the component by using orthographic and sometimes pictorial views.
- Giving actual sizes by dimensioning.
- Giving information about any special manufacturing processes required.

The design engineer should have a good understanding of projection methods, dimensioning methods and the manufacturing methods to be used.

This section introduces some basic guidelines and examples to help explain the general rules of dimensioning, based on BS 8888.

2.5.1 General rules.

- Dimensions should be placed on drawings so that they may be easily read.
- The drawing must include the minimum number of dimensions necessary to manufacture.
- A dimension should not be stated more than once unless it aids communication.
- It should not be necessary for the operator manufacturing the component to have to calculate any dimensions.

2.5.2 Types of dimension.

Types of dimensioning can be broadly classified as:

- Size dimensions. Used to describe heights, widths, diameters, etc.
- Location dimensions. Used to place various features of a component relative to each other, such as a hole centre line to a reference surface.
- **Mating** dimensions. Used for parts that fit together requiring a certain degree of accuracy.

2.5.3 Dimensioning conventions.

2.5.3.1. General.

Observe the dimensioning features shown for the plate in Figure 2.5a below. Note:

- parallel dimensions, indicating the size of the plate
- edges A and B are being used as the reference edges
- minimum number of dimensions required are specified
- use of description of 'plate 3mm thick', so that no side view is required
- evenly spaced dimension lines









Circles on engineering drawings are usually either spheres, holes or cylinders of some description. The dimension refers to the diameter, and the diameter symbol is \emptyset .



Holes equally spaced on a pitch circle can be dimensioned as shown below.



The \varnothing 40 dimension can also be referred to as the **PCD** or **Pitch Circle Diameter**.



2.5.3.5 Location dimensions:

Due to the nature of manufacturing, actual finished dimensions of manufactured components are never perfect. This has to be considered when dimensioning features that require accurate location. Inorder to enable accyrate measurement, such a feature is usually dimensioned from a reliable reference such as a machined surface. This reference is referred to as a **Datum**.



Figure 2.5b.

Figure 2.5b shows: **1)** A spigot located from two reference edges.

2) Two holes located from two reference edges.

3) The large hole located from two reference edges and the small hole from the center of the large hole.

The simple bearing bracket casting below shows both size and location dimensions.



Note that machined surfaces are specified using this British Standard machining symbol:



2.5.3.5 Surface finish:

Surface textures resulting from manufacturing processes consist of many complex peaks and valleys varying in height and spacing. The **Roughness value** of a surface is a measure of this surface quality. The table below gives some nominal values of roughness resulting from various common manufacturing processes.

If a particular surface finish is required you give clear instructions on the drawing using the British Standard machining symbol.

	_			_				_					-		-
Roughness number, N	12	11	10	9)	8	7	6		5	4	3	2	2	1
Roughness value, R_a (μ m)	50	25	12.	56.	33	2 1	.6	0.	8 0	4	0.2	0.	1 0.0	05 0	025
Super polishing											×				
Lapping															
Polishing														0.44.04	
Honing							50030	25	2002)¥					NADA)	2 82
Grinding						3433									1
Boring, turning		3		2/4						1.1.1	1000				1
Die casting				12.9.24	100	1000	1				T				
Reaming					3050										
Broaching					Sasta.		I								
Cold rolling											20				
Drawing						2.74									
Extruding			2		12-140 				2000 2000						
Milling		5							enera Mara	1	144				
Planing, shaping	23	2 (22):5							Kentige Kentige		2.5				
Drilling		1005	7				1.20	32	22433	Z					
Forging			6330			212	19419. 19	215							
Sawing	2	9 875		14 M		5202									
Hot rolling		8888 888				010100	20								
Sand casting		2000 (1969)		2022 2022											
Flame cutting	20 55	199381 64670													

2.6 Tolerances, limits and fits.

In order to ensure that assemblies function properly their component parts must **fit together** in a predictable way. As mentioned in section 2.5, no component can be manufactured to an exact size, so the designer has to decide on appropriate upper and lower limits for each dimension.

 Accurately toleranced dimensioned features usually take much more time to manufacture correctly and therefore can increase production costs significantly.
Good engineering practise finds the optimum balance between required accuracy for the function of the component and minimum cost of manufacture.

2.6.1 Dimension tolerances.

If a dimension is specified, in millimeters, as 10 ± 0.02 , the part will be acceptable if the dimension is manufactured to an actual size between 9.98 and 10.02 mm. Below are some examples of ways of defining such limits for a linear dimension.



To give you a feel for the magnitude of decimal values in mm, consider these facts:

The thickness of the paper this page is printed on is approximately **0.100 mm**.

Average human hair thickness is approximately **0.070 mm**.

The human eye cannot resolve a gap between two points smaller than about **0.020mm**, at a 20cm distance.

If you raise the temperature of a 100mm long block of steel by 10°C it will increase in length by approximately **0.020mm**.

2.6.2 General tolerancing.

General tolerance notes apply tolerances to all unspecified dimensions on a drawing. They can save time and help to make a drawing less cluttered. Examples are shown below in Figure xx.

TOLERANCE EXCEPT WHERE OTHERWISE STATED ± 0.5	TOLERANCE	TOLERANCES EXCEPT WHEN OTHERWISE STATED				
	SI	ZE	TOLERANCE			
		UP TO X	± 0.1			
	OVER X	UP TO XX	± 0.25			
	OVER XX	UP TO XXX	± 1			
	OVER XXX		±2			
		ON ANGLES	± 0.5°			
TOLERANCE ON CAST THICKNESSES ± 1 %	FOR TOLER	ANCES ON FOR	RGING 4			

Figure xx. Some examples of general tolerance notes.

2.6.3 Limits and fits for shafts and holes.

2.6.3.1 Basic size and shaft/hole tolerancing systems.

The **basic size** or **nominal size** is the size of shaft or hole that the designer specifies before applying the limits to it. There are two systems used for specifying shaft/hole tolerances:

Basic hole system:

Starts with the basic hole size and adjusts shaft size to fit.



Basic shaft system:

Starts with the basic shaft size and adjusts hole size to fit.



Because holes are usually made with standard tools such as drills and reamers, etc, the **basic hole system** tends to be preferred and will therefore be used here

SIZ

BASIC

The **fit** represents the tightness or looseness resulting from the application of tolerances to mating parts, e.g. shafts and holes. Fits are generally classified as one of the following:

Clearance fit:	Assemble/disassemble by hand. Creates running & sliding assemblies , ranging from loose low cost, to free-running high temperature change applications and accurate minimal play locations.
Transition fit:	Assembly usually requires press tooling or mechanical assistance of some kind. Creates close accuracy with little or no interference.
Interference fit:	Parts need to be forced or shrunk fitted together. Creates permanent assemblies that retain and locate themselves.

2.6.3.3 ISO limits and fits.

Fits have been standradised and can be taken directly from those tabulated in the BS 4500 standard, '**ISO limits and fits**.'

The BS 4500 standard refers to tolerance symbols made up with a letter followed by a number. The BS Data Sheet BS 4500A, as shown on the following two pages, shows a range of fits derived, using the hole basis, from the following tolerances:

Holes:	H11	H9	H8	H7					
Shafts:	c11	d10	e9	f7	g6	k6	n6	р6	s6

Remember:

- Capital letters always refer to holes, lower case always refer to shafts.
- The greater the number the greater or wider the tolerances.

The selection of a pair of these tolerances will give you the fit. The number of possible combinations is huge. BS 4500 helps to standardise this and offers a range of fits suitable for most engineering applications.

Examine an extract from the BS 4500 data sheet on page 4 & 5 and you will observe the general class of fit specified on the top row. A more detailed description of the fit is given on the bottom row.

See the table in section 2.6.4 for guidance on the selection of types of fit.

			A				Clearan	ce fits					
Hol			нп	Ĩ	2	I	,	H M	8		17 7771 5553		17
Diagr Sho	fts am to e for diameter		e II	d	10				7				
Nomin	al sizes	Tole	rance	Tole	rance	Tole	rance	Tole	rance	Tole	rance	Tole	rance
Over	То	нп	cll	H9	010	H9	69	H8	17	H7	g 6	H7	h6
mm	115103	0-001 mm	0 001 mm	0-001 mm	0 001 mm	0 001 mm	0.001 mm	0-001 mm	0-001 mm	0 001 mm	0-001 mm	0-001 mm	0-001 mm
\sim	3	+ 60	- 60 - 120	+ 25	- 20 - 60	+ 25	- 14 - 39	+ 14	- 6 - 16	+ 10	= 2	+ 10	- 6
3	6	+ 75	- 70 - 145	+ 30	- 30 - 78	+ 30	- 20 - 50	+ 18	- 10 - 22	+ 12	- 4 - 12	+ 12	- 8
6	10	+ 90	- 80 - 170	+ 16	- 40	+ 36	- 25 - 61	+ 22	- 13 - 28	+ 15	- 5 - 14	+ 15	- 9
10	18	+ 110	- 95 - 205	+ 43	- 50 - 120	+ 43	- 32 - 75	+ 27	- 16	+ 18	-6	+ 18	- 11
18	30	+ 130	- 110 - 240	+ 52	- 65	+ 52	- 40	+ 33	- 20	+ 21	= 7	+ 21	- 13
30	40	+ 160	- 120 - 280	+0	- 10	10	- 10	4 10	- 36	1.36		1.94	
40	50	+ 160	- 130 - 290	ů,	- 180	0	- 112	0	- 50	+ 0	- 25	10	- 16
50	65	+ 190	- 140	+ 74	100	1.32	40	194	10	1.10	10	1.10	
65	80	+ 190	- 150	10	- 220	+ /t	- 134	+ 40	- 60	+ 30	- 29	+ 50	- 19
80	100	+ 220	- 170	1.07	120	1.00						10.000	
100	120	+ 220	- 180	ŤŐ	- 260	+ 8/	= 159	+ 54	- 71	+ 55	- 12 - 34	+ 35	- 22
120	140	+ 250	- 200										
140	160	+ 250	- 210	+ 100	- 145	+ 100	- 84	+ 63	- 43	+ 40	- 14	+ 40	- 25
160	180	+ 250	- 230	v	- 305	0.0	- 185	0	~ 83	0	- 39	0	0
180	200	+ 290	- 240									-	
200	225	+ 290	- 260	+ 115	- 170	+ 115	- 100	+ 72	- 50	+ 46	- 15	+ 46	- 29
225	250	+ 290	- 280	0	- 355	0	- 215	0	- 96	0	- 44	0	0
250	280	+ 320	- 300	- 0.00	1,632	17/19/2017	11.5424	1000.0	2.25			1 2392	
280	315	+ 320	- 330	+ 130	- 190 - 400	+ 130	- 110 - 240	+ 81	- 56 - 108	+ 52	- 17 - 49	+ 52	- 32
315	355	+ 360	- 160		1.1								
355	400	+ 360	- 400	0 + 140	- 210 - 440	+ 140	- 125 - 265	+ 89	- 62 - 119	+ 57	- 18 - 54	+ 57	- 36
400	450	+ 400	- 440								1.040		
450	500	+ 400	- 840	+ 155	- 230 - 480	+ 155	- 135 - 290	+ 47	- 131	+ 63	- 20 - 60	+ 63	- 40
1.00	0.00	Slac	- 880 k fit	Loo	se fit	Easy	fit	Nor	mal fit	Clo	se fit	Sli	de fit

	Transi	tion fits							
H7 1777-1	k6	H7 [7/7/2]	n6 	н7 97772	P6	H7	<u>56</u>	Ho	
Tole	TARCE	Tole	TARCE	Tole	rance	Tole	Tance	Sho	uf ts
H7	k6	H7	n6	H7	p6	H7	50	Over	To
0 001 mm	0.001 mm	0 001 mm	0 001 mm	0 001 mm	0.001 mm	0 001 mm	0 001 mm	mm	mm
+ 10	+ 6 + 0	+ 10	+ 10	+ 10	+ 12 + 6	+ 10	+ 20 + 14	\sim	3
+ 12	+9	+ 12	+ 16 + 8	+ 12	+ 20 + 12	+ 12	+ 27 + 19	3	6
+ 15	+ 10	+ 15	+ 19 + 10	+ 15	+ 24 + 15	+ 15	+ 12 + 23	6	10
+ 18	+ 12	+ 18	+ 23 + 12	+ 18	+ 29 + 18	+ 18	+ 39 + 28	10	18
+ 21	+ 15 + 2	+ 21	+ 28 + 15	+ 21	+ 35 + 22	+ 21	+ 48	18	30
+ 25	+ 18	+ 35	+ 11	+ 35		+ 35	1.69	30	40
ő	+ 2	0	4 17	Ť Ô	+ 26	+ 0'	+ 43	40	50
+ 30	4.20	1. 30	4.10	3.30		+ 30	+ 72	50	65
0	‡ 2'	+ 50	7 20	+ 30	7 32	+ 30	+ 78	65	80
+ 16	+ 35	4.15	1.45	1.15	1.59	+ 35	+ 93 + 71	80	100
0	4 5'	0	723	+ 35	+ 37	+ 35	+ 101 + 79	100	120
						+ 40	+ 117 + 92	120	140
+ 40	+ 28	+ 40	+ 52	+ 40	+ 68	+ 40	+ 125	140	160
	124125	×			4.90	+ 40	+133 +108	160	180
						+ 46	+ 151 + 122	180	200
+ 46	+ 33	+ 46	+ 60	+ 46	+ 79	+ 46	+ 159 + 130	200	225
					+ 30	+ 46	+ 169	225	250
4.52	+ 16	10	+ 66	1.0	1.94	+ 52	+190	250	280
õ	+ 4	0	7.4	10	+ 56	+ 52	+ 202	280	315
+ 57	+ 40	+ 57	+ 71	+ 57	1.08	+ 57	+ 226	315	355
Ö	+ 4	0	4 37	0	+ 62	+ 57	+ 244	355	400
+ 61	115	+ 61	1 80	4.61	1.104	+ 63	+ 272	400	450
0	+3	0	+ 40	0	+ 68	+ 61	+ 292	450	500
Push fit		Driv	e fit	Pre	ss fit	For	ce fit	1	1000

Consider an example of a shaft and a housing used in a linkage:

Type of fit:	'Normal' clearance fit
Basic or Nominal size:	∕Ø 40 mm

We will determine the actual working limits, the range of allowable sizes, for the shaft and the hole in the housing.

Look along the bottom of the ISO Fits Data Sheet 4500A and locate 'Normal Fit'. We will use this pair of columns to extract our tolerances.

The tolerances indicated are:	1 st column	H8	for the hole	(upper case H)	
	2 nd column	f7	for the shaft	(lower case f)	

The actual tolerances depend upon the basic, or nominal, diameter as well as the class of fit. So, locate 40mm in the left hand **Nominal Sizes** column. Either the **30 - 40** or **40 - 50** range is acceptable in this case. Read across and note the tolerance values for the hole and the shaft, as shown below.



For the hole diameter we have a tolerance of: +0.039mm -0.000mm

For the shaft diameter we have a tolerance of: -0.025mm -0.050mm

These tolerance values are simply added to the nominal size to obtain the actual allowable sizes.

Note that this is a clearance fit. As long as the hole and shaft are manufactured within the specified tolerances the hole will **always** be either slightly oversize or spot on the nominal size and the shaft will **always** be slightly undersize. This ensures that there will **always** be a free clearance fit.

These tolerances may be expressed on a drawing in several ways:

1) Simply as the nominal size with the tolerance class.

This is not always preferred as the machine operator has to calculate the working limits.





Ø 40 н8

(40.039) (40.000)

2) The nominal size with the tolerance class as above with the calculated working limits included.







3) The calculated working limits only.

More clearance			→ More intertence								
	Description	Loose running tit for wide commercial tolerances or allowances on external members	Free running fit not for use where accuracy is essential, but good for large temperature variations, high running speeds, or heavy journal pressures	Close running fit for running on accurate machines and for accurate location at moderate speeds and journal pressures	Sliding fit not intended to run freely but to move and turn freely and locate accurately	Locational clearance fit provides snug fit for locating stationary parts but can be freely assembled and disassembled	Locational transisition fit for accurate location; a compromise between clearance and interference	Locational transition fit for more accurate location where greater interference is permissible	Locational interference fit for parts requiring rigidity and alignment with prime accuracy of location but without special bore pressure requirements	Medium drive fit for ordinary steel parts or shrink fits on light sections; the tightest fit usable with cast iron.	Force fit suitable for parts that can be highly stressed or for shrink fits where the heavy pressing forces required are impractical
rmbol	Shaft Basis	C11/h11	D9/h9	F8/h7	G7/hó	H7/h6	K7/h6	N7/h6	P7/h6	S7/h6	U7/h6
ISO SY	Hole Basis	H11/c11	6P/6H	H8/f7	H7/g6	H7/h6	Н7/К6	H7/n6	H7/p6*	H7/s6	H7/u6
← Clearance tits →					$\leftarrow \text{Interference tits} \rightarrow \leftarrow \text{Transition tits} \rightarrow$						

Engineering drawing & design

2.7 Assembly drawings.

Assembly drawings can be used to:

- Name, identify, describe and quantify all of the components making up the assembly.
- Clearly show how all of the components fit together.
- Indicate all of the required fasteners.
- Record any special assembly instructions.
- Record any other relevant information.

Here is an example:



Note the use of sections, item numbers neatly layed out and the parts list.